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COUPLER WITH IMPROVED PIN LOCK

Cross-Reference to Related Application

This application claims priority from and hereby expressly incorporates by reference U.S. provisional application no. 60/254,711 filed December 11, 2000.

Background of the Invention

5 The present invention relates to an improved quick-coupler for attaching a bucket or other implement to an excavator, backhoe, tractor, or other machine or apparatus having a boom/arm/dipper-stick to which a bucket or other implement (e.g., a shear, air-hammer, magnet, etc.) is operatively secured.

Quick-couplers are well-known and in widespread use. One such
10 coupler is sold under the trademark Slide-Loc® by JRB Company, Inc., Akron, Ohio, USA and is shown in FIGURE 1. The coupler body 134 is pinned (by pins 137,138) to the distal end 125 of the arm 124 and the distal end 128b of the control link 128. The coupler 134, in turn, is adapted for selective and convenient interconnection with an associated implement, such as a bucket, shear, etc. Specifically, the
15 coupler 134 includes first and second recesses 140,150 adapted for respective direct receipt of the first and second implement pins 152,154 without these pins being removed from the associated implement to which they are connected. A wedge lock member 160 is selectively extensible under force of a fluid cylinder 162 (or manually) into the second recess 150 to at least partially obstruct the recess 150

and, thus, capture the second pin 154 therein. To decouple the bucket or other implement from the coupler 134, the wedge lock member 160 must first be retracted to a position where it does not obstruct the mouth of the recess 150. The first recess 140 is formed so that, when the second pin 154 is captured in the second recess 150 by the lock 160, the first pin 152 cannot escape from the first recess 140. This type of quick coupler is highly effective, safe and has enjoyed widespread commercial success.

Couplers such as that shown in FIGURE 1 include means for preventing the accidental or inadvertent retraction of the wedge lock member 160 that could result in unintended escape of the second pin 154 from the second recess 150. However, a need has been identified for a new and improved supplemental lock mechanism that does not require an operator to exit the cab of the machine being operated, that is resistant to damage and tampering and that is resistant to operator error.

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Summary of the Invention

In accordance with a first aspect of the present invention, a coupler includes a body having first and second recesses adapted for respectively receiving first and second associated pins of an associated implement. A first lock member is movably secured to the body and is movable between a retracted position one or more extended positions. The first lock member at least partially obstructs the second recess when it is moved to an extended position so as to capture a second associated pin in the second recess. An actuator is operably coupled to the first

lock member and is operable to move the first lock member between its retracted and extended positions. A supplemental lock member is movably secured to the coupler body and is movable between an unlocked position and one or more locked positions. When the supplemental lock member is in its locked position, it engages
5 and prevents movement of the first lock member from the extended position to the retracted position. The supplemental lock member is movable from its locked position to its unlocked position in response to movement of said coupler body to a select angular position relative to the associated arm.

In accordance with another aspect of the present invention, a coupler
10 includes a body with first and second pin-receiving recesses. A main lock member is selectively movable between an extended position where the main lock member at least partially obstructs one of the recesses and a retracted position. A supplemental lock member is connected to the body and is movable between a locked position, where the supplemental lock member blocks movement of the main
15 lock member from the extended position to the retracted position, and an unlocked position. The supplemental lock member includes a portion adapted to engage an associated arm to which the body is connected only when the coupler body is at least substantially curled relative to the associated arm. When the coupler body is at least substantially fully curled, the supplement lock member moves from the
20 locked position to the unlocked position.

In accordance with another aspect of the present invention, a coupler includes a body having first and second pin-receiving recesses and that is adapted for pivotable connection to an associated arm. A main lock member is selectively

movable to a locked position where it at least partially obstructs one of the recesses to capture an associated pin therein. The coupler further includes means for preventing movement of the main lock member out its locked position unless the body of the coupler is at least substantially fully curled relative to the associated arm.

One advantage of the present invention resides in the provision of a new and improved coupler that is easy to use and that substantially enhances safety and productivity by minimizing the chance for operator error.

Another advantage of the present invention is found in the provision of a coupler that provides an operator with at least one visual indication that the coupler is in a "locked" condition.

~~A further advantage of the present invention resides in the provision of a coupler with a supplemental pin-lock mechanism that is very robust and resistant to damage and malfunction in harsh environments.~~

Still another advantage of the present invention is found in the provision of a coupler with improved pin lock wherein an operator need not exit the cab of the excavator or other machine to operate the supplemental lock.

A still further advantage of the present invention resides in the provision of a coupler with improved pin lock wherein a bucket or other implement cannot be de-coupled without the coupler first being at least substantially fully curled relative to the arm to which the coupler is attached.

Another advantage of the present invention resides in the provision of a coupler with improved pin lock wherein electronic proximity sensors are used

to provide an operator with information concerning the state of the coupler.

Still other benefits and advantages of the present invention will be apparent to those of ordinary skill in the art to which the invention pertains upon reading and understanding this specification.

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Brief Description of the Drawings

The invention comprises various components and arrangements of components, preferred embodiments of which are illustrated in the accompanying drawings that form a part hereof and wherein:

FIGURE 1 (prior art) illustrates a conventional coupler connected to
10 an associated arm;

FIGURE 1A illustrates a coupler formed in accordance with the present invention in a sectional view taken along line A-A of FIGURE 3 and as connected to an associated arm, with the coupler in a first locked condition;

FIGURE 1B is similar to FIGURE 1A but illustrates the coupler in a
15 second locked condition;

FIGURE 2 illustrates the coupler of FIGURES 1A and 1B in an unlocked condition; and,

FIGURE 3 is a simplified top plan view of a coupler formed in accordance with the present invention.

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Detailed Description of Preferred Embodiments

A coupler C formed in accordance with the present invention is shown

in FIGURES 1A, 1B, 2 and 3. The coupler C is pivotally connected by pin-on arrangement to an associated arm A of an excavator at a pivot point P1 in a conventional manner. The coupler C is also adapted for pivotable connection by pin-on arrangement to a control link member (not shown) or like control member at point P2.

The coupler C defines or otherwise includes a first recess R1 adapted for receipt of a first pin 10 of a bucket or other associated implement and a second recess R2 adapted for receipt of a second pin 12 of the bucket or other associated implement. As is well known, owing to the direction in which the recesses respectively open, the first pin 10 of the associated implement must be inserted into the recess R1 before the second pin 12 of the bucket or other implement is received in the second recess R2.

As with known couplers, the coupler C includes a wedge W or other main or first lock member that is adapted for sliding linear movement as indicated by the arrows L1, L2 between a retracted, unlocked position (FIGURE 2) and one or more extended, locked positions (FIGURES 1A and 1B). As illustrated in FIGURES 1A and 1B, when the wedge W is moved to either the first locked position (FIGURE 1A) or the second locked position (FIGURE 1B), the wedge W is extended in the direction L1 and at least partially obstructs the recess R2 so as to capture the second pin 12 of the associated bucket or other implement in the recess R2. In the unlocked position as shown in FIGURE 2, the wedge W is retracted relative to the recess in a direction L2 (opposite the direction L1) at least an amount sufficient to allow the second pin 12 of the associated bucket or other

implement to move freely into and out of the recess R2 for coupling/decoupling operations.

Those of ordinary skill in the art will recognize that the recess R2 is conformed and dimensioned to accommodate multiple pin diameters and centers.

5 The ability of the wedge lock member W to be moved to multiple locked (extended) positions as described ensures that the second pin 12 located in the recess R2 will be tightly held in the recess with significant contact between the outer surface of the pin 12 and the surface defining the recess to increase safety and minimize wear. Furthermore, safety is also enhanced because the ability of the recess R2 and
10 wedge lock member W to cooperate and accommodate multiple diameters and centers for the pin 12 helps to ensure that the first pin 10 is fully and tightly seated in the first recess R1 rather than only partially seated therein. Furthermore, as the recess R2, pin 12 and/or wedge lock member W wear, the wedge lock member W can be extended further to ensure that the pin 12 can still be tightly held in the
15 recess R2.

The coupler C comprises means for extending and retracting the wedge W as described. Preferably, a hydraulic or other fluid cylinder or other actuator F is provided for extending and retracting the wedge W, although other structure and/or means such as mechanical and electro-mechanical (e.g., solenoid)
20 means are contemplated and can be used without departing from the overall scope and intent of the present invention. Also, while the wedge lock member W is slidably movable between its extended or retracted positions, it is not intended that the invention be limited to this arrangement. The illustrated fluid cylinder includes

a rod R that extends and retracts and that is operably connected to the wedge W.

Referring now to FIGURE 3, wherein the fluid cylinder F and other components are not shown for clarity, it can be seen that the coupler C comprises a body including first and second parallel, spaced-apart and interconnected sidewalls S1,S2 that define an open channel T therebetween. The pivot points P1,P2 are provided by aligned bores A1,A2 and A3,A4, respectively, defined in the sidewalls S1,S2. Thus, the arm A (FIGURES 1A, 1B and 2) is received in the channel T and connected to the coupler C by a pin-on connection at the bores A1,A2. Likewise, the bucket link or other control link member that is connected at the pivot point P2 is received in the channel T and operatively connected to the coupler by a pin-on connection at the bores A3,A4. A cross-pin B (shown only partially in FIGURES 3 for clarity) or a like member extends between the sidewall S1,S2. In the illustrated embodiment, the cross-pin is received in aligned apertures A5,A6 and is welded or otherwise fixedly secured to the sidewall S1,S2 while spanning the channel T. The fluid cylinder F is preferably fixedly secured to the coupler C by connection of a rear portion F1 thereof (i.e., a portion located opposite the rod R) to the cross-pin B. This arrangement helps to prevent exertion of undesired stresses on the body of the cylinder F as can cause malfunction of the cylinder.

The coupler C further comprises a supplemental lock arm E that is movably secured to the coupler body and adapted for movement between at least one locked position and an unlocked position. With specific reference to the illustrated embodiment, the lock arm E includes an aperture F defined therethrough,

and the arm is secured to the coupler C by passage of the cross-pin B through the aperture F. Alternatively, a bolt or other fastener or any other suitable arrangement is used to secure the lock arm E movably to the coupler C. The lock arm E is adapted for limited pivotal movement about the cross-pin B between a first locked position (FIGURE 1A), a second locked position (FIGURE 1B) and an unlocked position (FIGURE 2). A torsion spring G or other biasing means normally biases and urges the lock arm E toward the second locked position at all times as indicated by the arrow G1.

The lock arm E includes first and second opposite ends E1,E2. The first end E1 is defined to include first and second stop surfaces E1a,E1b, wherein the first stop surface E1a is recessed relative to the second stop surface E1b so that a third stop surface E1c (FIGURE 2) is also defined. When the wedge lock member W is moved to its first extended position as shown in FIGURE 1A, the spring G urges the lock arm E in the direction G1 so that the arm E is held in the first locked position as also shown in FIGURE 1A, with the first stop surface E1a of the lock arm E in abutment with or adjacent a rear portion W1 (FIGURE 3) of the wedge W and the third stop surface E1c contacting an the wedge W. When the lock arm E is located in its first locked position as described, the first stop surface E1a of the arm prevents movement of the wedge lock member W from the first locked position to the unlocked position (i.e., the stop surface E1a of the arm prevents movement of the wedge W in the direction L2 from its first locked position to the retracted position). Similarly, when the wedge lock member W is moved to its second locked position as shown in FIGURE 1B, the lock arm E moves in the

direction G1 under the force of the spring G to its second locked position wherein the second stop surface E1b is in abutment with or adjacent the rear portion W1 of the wedge lock member W. When the lock arm E is so positioned, the stop surface E1b prevents movement of the wedge lock member W from its second locked position toward the unlocked position.

The coupler C includes or defines a stop H that prevents movement of the lock arm in the direction G1 beyond the second locked position. Most preferably, this stop H is defined by round stock or otherwise conformed to resist build-up of dirt and debris that could prevent movement of the lock arm E into its first and second locked positions. Also, as shown herein, the wedge lock member W moves over and clears debris off of the stop H each time the wedge lock member W is retracted. It is also most preferred that the first and second stop faces E1a, E1b of the lock arm E and/or the wedge portion W1 be conformed with sloped faces or otherwise so that contact between the wedge portion W1 and either stop face E1a, E1b upon attempted movement of the wedge W in the direction L2 results in the lock arm E being urged in the direction G1 (toward the stop H) rather than the opposite direction so as to ensure that wedge W, itself, cannot move the lock arm E from one of its first and second locked positions to its unlocked position.

The lock arm E is selectively movable to its unlocked position in response to pivoting movement of the coupler C relative to the arm A to a select angular position. Preferably, as shown in FIGURE 2, pivoting movement of the coupler C on the arc P from its uncurled position (FIGURES 1A, 1B) to its fully or at least substantially fully curled position (FIGURE 2) causes the second end E2 of

the lock arm E to contact the associated arm A of the excavator or other machine so that the lock arm E is pivoted from either of its first or second locked positions to its unlocked position (FIGURE 2) against the force of the spring G. For purposes of this document, the fully curled or at least substantially fully curled position means

5 that the coupler C is pivoted sufficiently so that the first pin 10 of an associated bucket or other implement will not exit the first recess R1 and will be held therein by gravity even if the wedge lock member W is fully retracted and the second pin 12 is able to escape the second recess R2. This position may be obtained, e.g., by full extension of the cylinder to which the control link is operably connected. When the

10 lock arm E is in its unlocked position (FIGURE 2), the end E1 thereof is moved away from the rear portion W1 of the wedge W so that the wedge can be retracted freely in the direction L2 from either its first or second locked position to its unlocked position as shown in FIGURE 2. When the wedge lock member W is retracted as shown in FIGURE 2, the first end E1 of the lock arm E will rest on the wedge W

15 (even if the coupler C is uncurled) until the wedge is once again extended to one of its locked positions, at which time the spring G will move the arm E to its corresponding locked position as described above. Thus, the supplemental lock member E provides means for preventing retraction of the main lock member W unless the coupler body is at least substantially fully curled relative to the associated

20 arm A to which the coupler body is pivotably connected.

As illustrated herein, the second end E2 of the lock arm E preferably includes an adjustable stem J connected thereto by pins, bolts, screws or other fasteners K1, K2 and that contacts the associated arm A. The stem J can be used

to extend the length of the arm E and/or the angle between the stem J and the arm E can be adjusted to vary the contact position of the stem J with the associated arm A to facilitate the coupler being used in connection with arms A of different dimensions and shapes.

5 When the coupler C is operably coupled to an arm A of an associated excavator or other machine, the second end E2 of the lock arm, with or without the stem J, provides a visual indicator to an operator of the machine concerning the position of the lock arm E. In particular, when the lock arm E is located in its unlocked position, it is located closely adjacent the body of the coupler C in the
10 region of the first recess R1. This location of the second end E2 of the lock arm will be readily apparent to an operator and will notify the operator that the lock arm E has not moved into one of its locked positions under the force of the spring G as intended due to the presence of dirt or debris or some malfunction. On the other hand, as shown in FIGURES 1A and 1B, when the lock arm E is located in one of
15 its locked positions, the second end E2 thereof is spaced from the region of the coupler body defining the recess R1. Accordingly, the visual indicator provided by the second end E2 of the arm E (and the stem J if present) minimizes the chance that an operator will operate the bucket or other implement connected to the coupler when the lock arm is not in one of its locked positions. The second end E2 of the
20 arm and the stem J are preferably painted with bright, highly visible colors (e.g., red, pink, etc.) that contrast with the color of the arm A and the coupler body to increase visibility.

Furthermore, it should be noted that the lock arm E is intended to

provide a supplemental lock function only and is not intended to be the only lock or the primary lock used to hold the wedge lock member W in position. Thus, in the event the cylinder F or other actuator means for moving/holding the wedge lock member W to/in one of its extended positions is malfunctioning, the coupler C should not be used with only the supplemental lock arm E inhibiting movement of the wedge to its retracted position.

~~It is most preferred that the coupler C include a first proximity sensor PS1 connected thereto and adapted to sense the presence of the second pin of the bucket or other implement when the second pin is fully seated in the second recess R2. The proximity sensor is electrically connected to an indicator light and/or buzzer located in the cab of the excavator. Upon the second pin being fully seated in the recess R2, the operator is notified by the light and/or buzzer. This prevents the operator from extending the wedge W to obstruct the recess R2 until the second pin of the bucket or other implement is fully received in the recess R2.~~

It is preferred that a second proximity sensor PS2 be provided as a part of the coupler C. Specifically, it is preferred that the second proximity sensor PS2 be connected to the lock arm E or the stop H and sense contact or adjacency of the lock arm E with the stop H. Here, again, the proximity sensor PS2 is electrically connected to an indicator light and/or buzzer in the operator can and the light and/or buzzer is operated when the arm E contacts the stop H. In this manner, the operator can be certain that the lock arm E has moved to its first or second locked position under the force of the spring G and was not prevented from doing so by dirt, debris, or malfunction.

The invention has been described with reference to preferred embodiments. Modifications and alterations will occur to those of ordinary skill in the art to which the invention pertains. It is intended that the invention be construed as encompassing all such modifications and alterations insofar as they fall within the scope of the following claims as construed literally and/or according to the doctrine of equivalents.
